The logo of AIKTC (Amal Jyoti Kalsekar Institute of Technology) is a circular emblem. It features a central illustration of a classical building with a dome and arches. The text 'AMAL JYOTI KALSEKAR' is written in a semi-circle at the top, with 'ENGINEERING & TECHNOLOGY' below it. On the right side, it says 'TECHNICAL CAMPUS' and 'ARCHITECTURE'. At the bottom, it reads 'NAVI MUMBAI - INDIA'. The acronym 'AIKTC' is prominently displayed in the center of the emblem.

MODULE 05

DESIGN OF PUMP

- 1) Gear Pump
- 2) Centrifugal Pump



01) Gear Pump

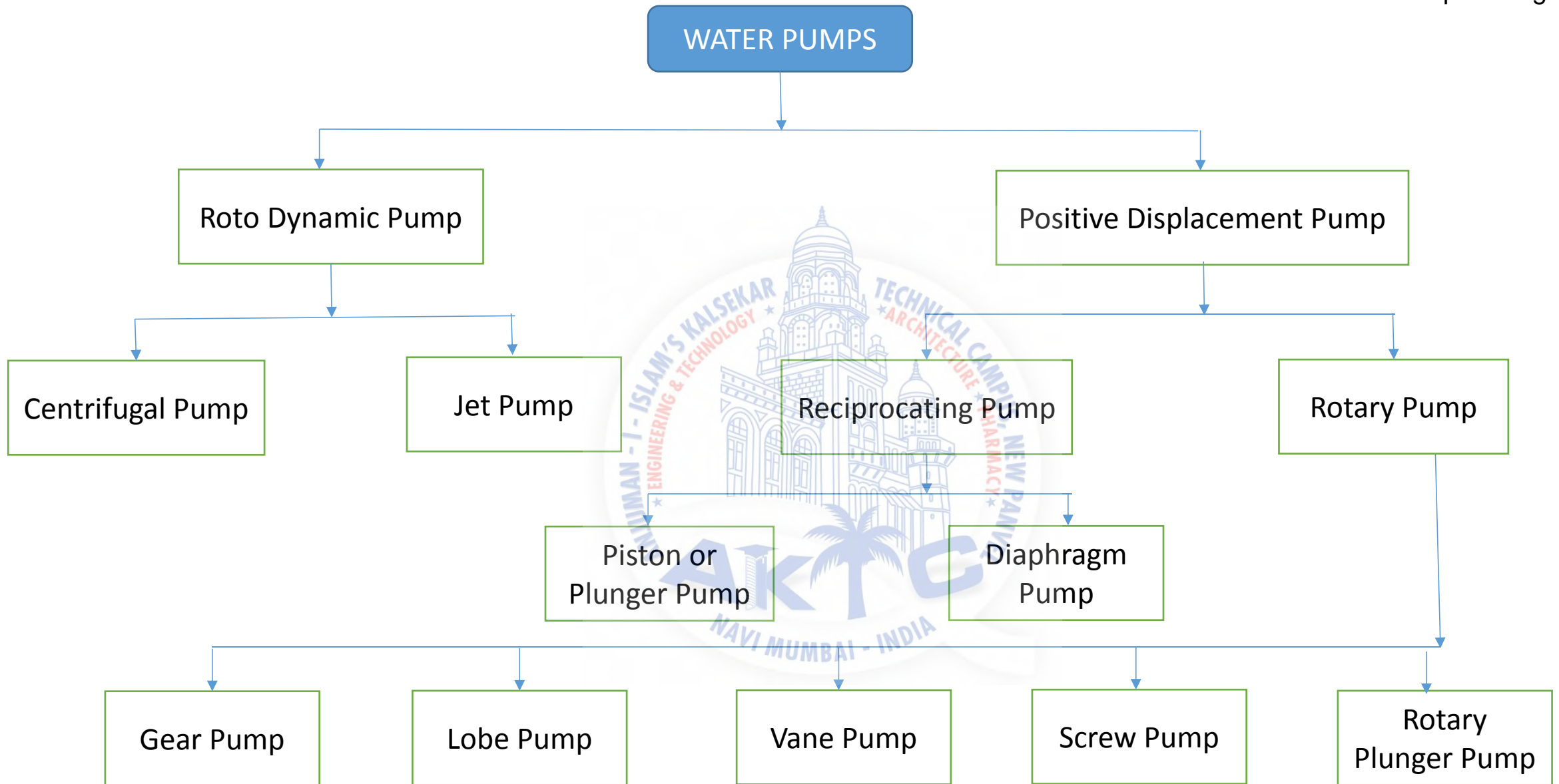
- Suction and Delivery Pipe
- Motor Selection
- Gear Design
- Shaft Design
- Bearing Selection
- Casting and Bolts Design



02) Centrifugal Pump

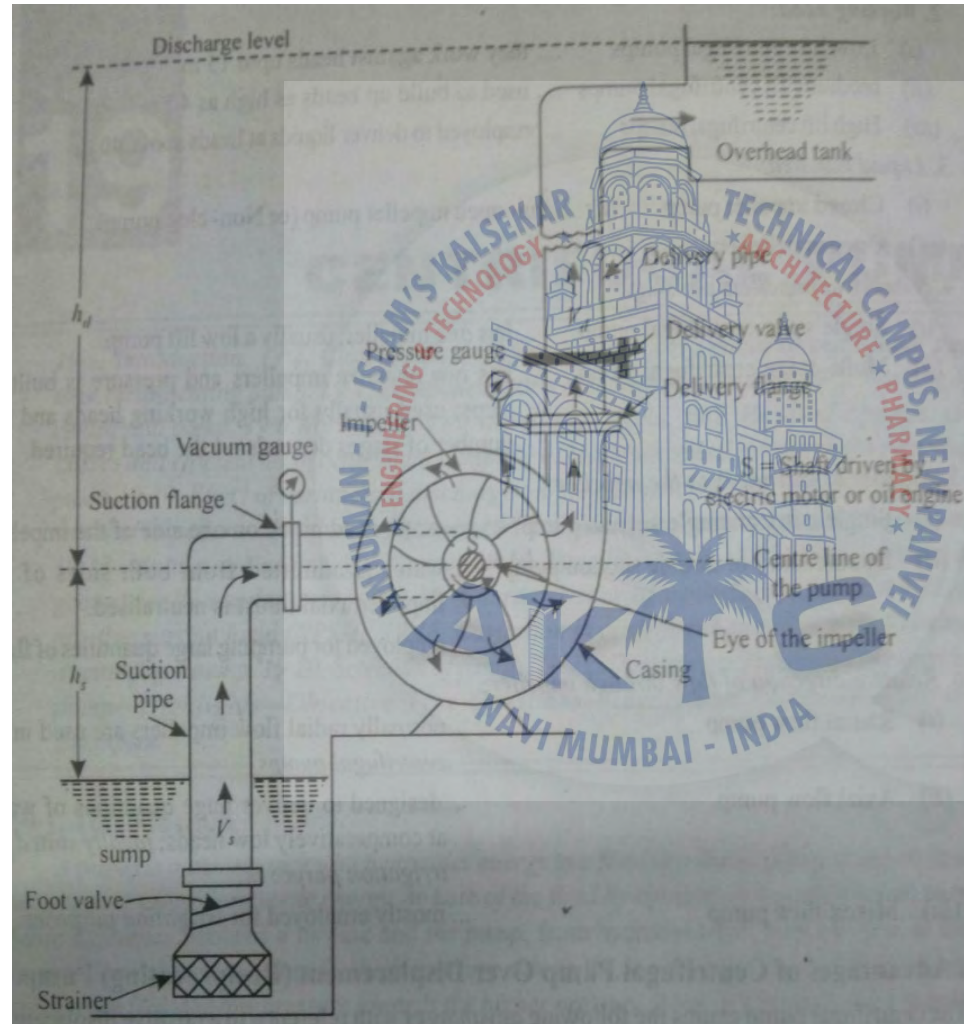
- Suction and Delivery Pipe
- **Motor Selection**
- Design of Impeller
- **Impeller Shaft**
- Design of Volute casing





02) Design of Centrifugal Pump

Step 01] General arrangements:-



STEP 02] ASSUMPTIONS:-

- Volumetric efficiency:
- Mechanical efficiency:
- Velocity in section pipe;

$$V_s = 1.5 \text{ to } 3 \text{ m/s}$$

- Velocity in delivery:

$$V_d = 4 \text{ to } 6 \text{ m/s}$$

STEP 03] DESIGN OF SUCTION AND DELIVERY PIPES:-

- Theoretical discharge:

$$\eta_{vol} = \frac{Q_{act}}{Q_{th.}}$$

- Design of suction pipe:

$$Q_{th.} = A_s \times V_s$$

- Design of delivery pipe:

$$Q_{th.} = A_s \times V_s$$



STEP 04] CALCULATION OF MANOMETRIC HEAD:-

- Calculate the manometric head by considering losses in suction and delivery pipes.

STEP 05] MOTOR SELECTION:-

- Motor power:

$$Power = \frac{\omega \cdot Q_{th} \cdot H_{man}}{\text{overall efficiency}}$$

- Speed :

Have to select from design data book.



STEP 06] DESIGN OF IMPELLER:-

Calculate thickness, width and number of blades.

- Eye Diameter=Suction pipe dia.
- Blade speed at inlet (u_1):

$$u_1 = \frac{\pi D_1 N}{60}$$

- Blade speed at outlet (u_2):

$$K_u = \frac{u_2}{\sqrt{2 g H_{man}}}$$

- Dia. of impeller at out let:

$$u_1 = \frac{\pi D_1 N}{60}$$

- Number of blade (Z):

$$Z = 6.5 \left(\frac{D_1 + D_2}{D_1 - D_2} \right) \sin \left(\frac{\alpha + \beta}{2} \right)$$

α & $\beta \rightarrow$ can be calculate by using inlet and outlet velocity diagram



- Width and thickness at inlet and outlet:

$$Q_{th} = K \pi D_1 b_1 V f_1$$

$$Q_{th} = K \pi D_2 b_2 V f_2$$

Take $K = 0.92$

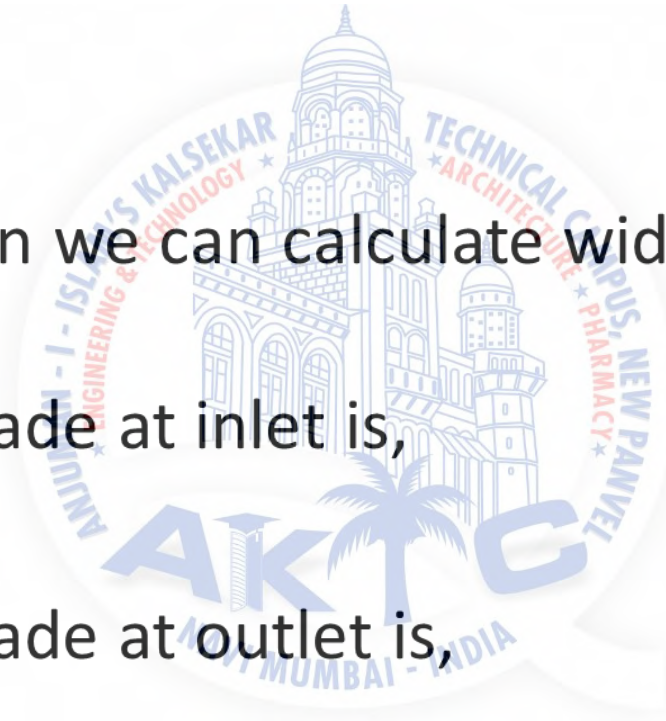
From above equation we can calculate width b_1 and b_2 at inlet and outlet respectively.

- Assuming thickness of blade at inlet is,

$$t_1 = 3 \text{ to } 5 \text{ mm}$$

- Assuming thickness of blade at outlet is,

$$t_2 = 6 \text{ to } 8 \text{ mm}$$



STEP 07] DESIGN OF IMPELLER SHAFT:

- Calculate torque:

$$T = \frac{P \times 60}{2\pi N}$$

- Calculate bending moment:

Calculate static and Dynamic load of impeller.

i) Static load of impeller (F_s): weight of impeller and weight of front and back shroud.

ii) Dynamic load of impeller (F_d): centrifugal force due to unbalance mass of impeller. [Unbalance mass have to take 10% of the whole mass]

iii) bending load on shaft $F = F_s + F_d$

- Calculate equivalent torque:
- Calculate shaft diameter by using equivalent torque.

Step 08] Selection of Bearing:

- Radial load on bearing:
 $F_r =$ Support reaction at shaft.
- Axial load on bearing:
- Bearing speed(N):
 $N =$ Speed of Gear
- Life of bearing:
- Equivalent load (P_{eq}):
 $P_{eq} = 1.5 \times F_r$
- Dynamic load C:
 $C = (L)^{1/k} \cdot P$
- Select the bearing from PSG by using dynamic load carrying capacity. And check for life required.



STEP 09] DESIGN OF CASING:-

- Design volute casing, have to calculate radius of casing at different angle:

$$R_{\theta} = R_2 + 5\text{mm} + d_{\theta}$$

$R_2 \rightarrow$ outside radius of impeller

$$d_{\theta} = d_{th} \sqrt{\frac{\theta}{360-2\theta}}$$

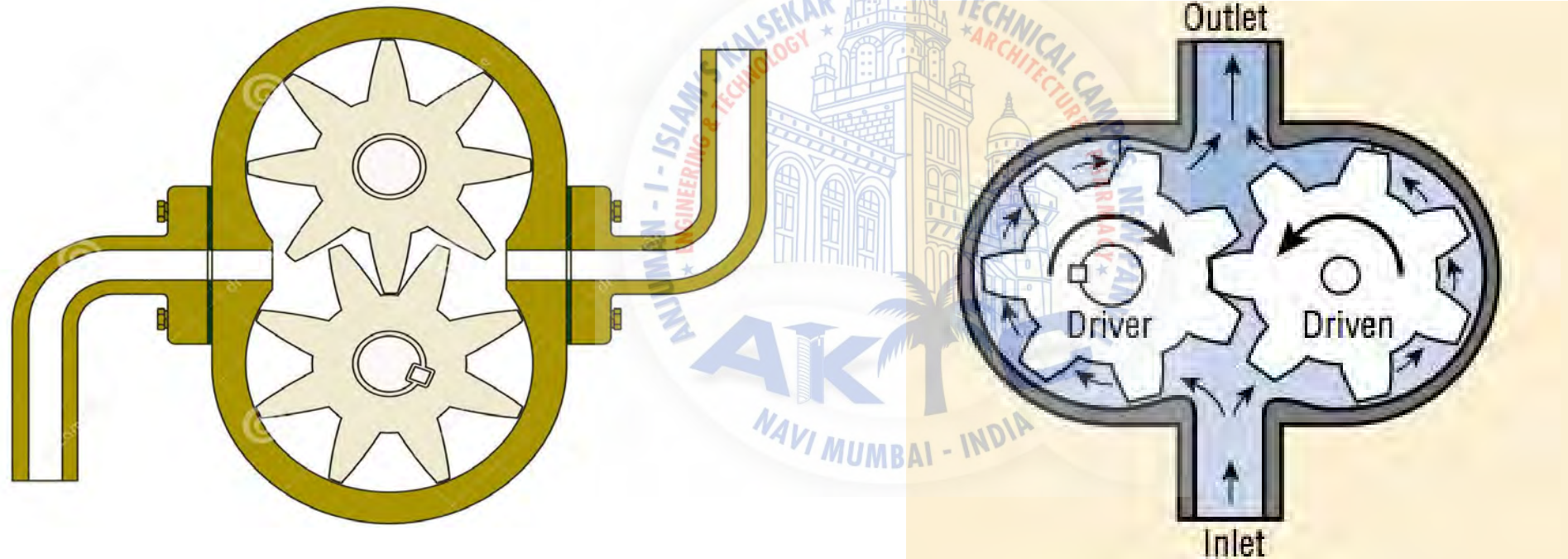
$$Q_{th} = \frac{\pi}{4} d_{th}^2 V_{th}$$

$$V_{th} = 0.25 u_2$$

STEP 10] DESIGN OF FLANGE AND FASTENER:-

01) Design of Gear Pump

Step 01] constructions:-



STEP 02] ASSUMPTIONS:-

- Volumetric efficiency:
- Mechanical efficiency:
- Velocity in section pipe;

$$V_s = 1.5 \text{ to } 3 \text{ m/s}$$

- Velocity in delivery:

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STEP 03] DESIGN OF SUCTION AND DELIVERY PIPES:-

- Theoretical discharge:

$$\eta_{vol} = \frac{Q_{act}}{Q_{th.}}$$

- Design of suction pipe:

$$Q_{th.} = A_s \times V_s$$

- Design of delivery pipe:

$$Q_{th.} = A_s \times V_s$$



STEP 04] CALCULATION OF MANOMETRIC HEAD:

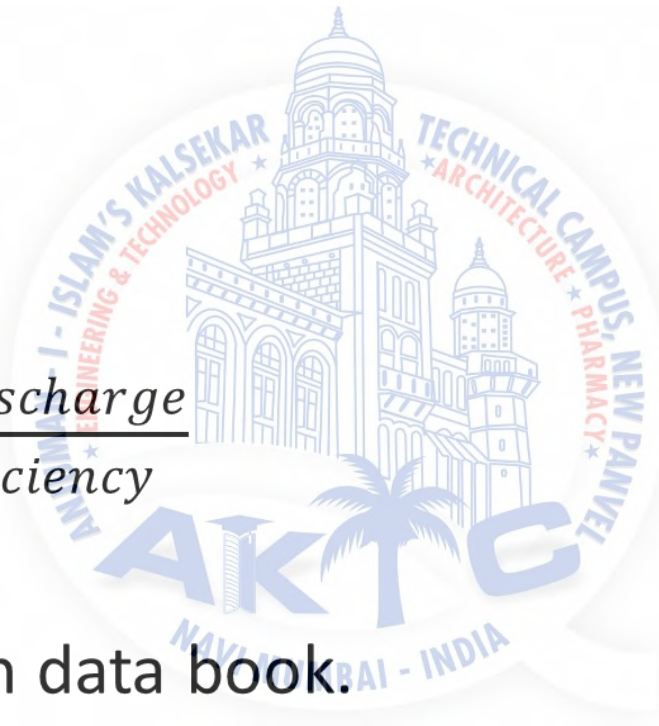
- Calculate the manometric head by considering losses in suction and delivery pipes.

Step 05] Motor Selection:

- Motor power:

$$Power = \frac{pressure \times discharge}{overall\ efficiency}$$

- Speed :
- Have to select from design data book.



STEP 06] DESIGN OF SPUR GEAR:

- Speed and number of teeth of gear and pinion:
consider number of teeth for gear and pinion: 12 to 14.
- Material for gear:
Select from [PSG 1.9]
and calculate bending and contact stress [PSG 8.18 & 8.16]
- Calculate module (m):
$$Q_{th} = 2 \pi b m^2 z N$$
- Checking Lewis dynamic load: [PSG 8.50]
- Checking Buckingham's dynamic load: [PSG 8.51]
- Calculate principle dimensions of gear. [PSG 8.22]
- Design the shaft and hub to support the gears.

STEP 07] SELECTION OF BEARING:

- Radial load on bearing:

F_r = Support reaction at Gear shaft.

- Bearing speed(N):

N = Speed of Gear

- Life of bearing:

- Equivalent load (P_{eq}):

$P_{eq} = 1.5 \times F_r$

- Dynamic load C:

$C = (L)^{1/k} \cdot P$

Select the bearing from PSG by using dynamic load carrying capacity.

STEP 08] DESIGN OF CASING:

- Thickness of casing:

$$\frac{t}{r_1} = \left[\frac{1 + \frac{P_1}{[\sigma]}}{1 + \frac{P_1}{[\sigma]}} \right]^{1/2} - 1 \quad \dots\dots\dots [\text{PSG 6.15}]$$

Step 09] Design of fastener:

Step 10] Design of Coupling:

